

Monitoring Heat Rates as a Troubleshooting Tool

GMRC – Engine Analyzer &
Reliability Workshop
July 27 – 29, 2011
Loews Vanderbilt
Nashville, Tennessee

Introduction

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Lead Engineer

Columbia Gas Transmission

NiSource

- Columbia Gas of Pennsylvania (CPA)
- Columbia Gas Transmission (TCO)
- Columbia Gulf Transmission (CGT)
- Crossroads
- Hardy County Storage
- Midstream
- Millennium

NiSource



NiSource

	CPA	TCO	CGT	Crossroads	Hardy Storage	Midstream	Millennium	Totals
Compressor Stations	1	90	11	1	1	1	2	107
Total Horsepower	800	620,209	470,238	3000	7100	5280	19,020	1,125,647
Compressor Units	1	320	77	1	2	4	4	409
Reciprocating Units	1	232	53	0	2	4	3	295
Turbine Units	0	72	22	0	0	0	1	95
Electric Motor Units	0	16	2	1	0	0	0	19

Why on-line monitoring ?

- Infrastructure exists
- Programming tools exist
- Need to verify torque control programming
- Need to verify horsepower
- Need to verify surge control programming
- Need supporting tool for verifying the health of engines & compressors
- Need a tool for indicating trouble spots

Information – Reciprocating Engines & Compressors

- Horsepower
- Heat Rate / Fuel Consumption
- Throughput
- Unloading step
- Gas compressor discharge temperature
- Rod load
- Degrees of pin reversal

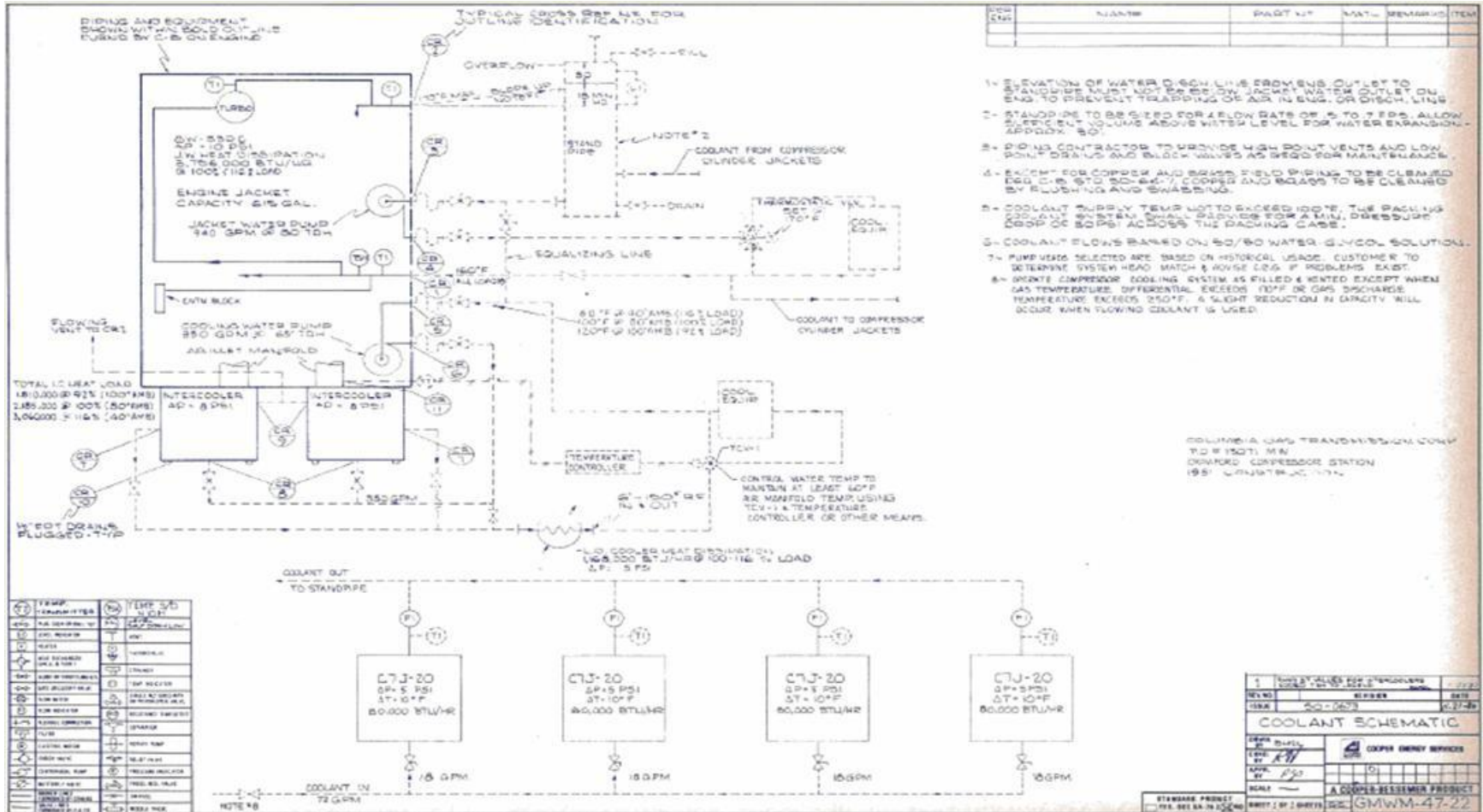
Information – Gas Turbine & Centrifugal Compressor

- Horsepower
- Heat Rate / Fuel Consumption
- Throughput
- Gas Compressor Discharge Temperature
- Surge Margin / Turndown
- Air compressor discharge pressure (Pcd)
- T5 (Solar)
- Exhaust Temperature (T7 – Solar)

Heat Rate

- Can be defined as the rate of fuel energy needed to produce one (1) unit of power.
- In the 'perfect world,' an engine would have a heat rate of 2545 BTU/BHP-hr.
- Is another way of representing thermal efficiency.

Coolant Schematic



REV.	DATE	NAME	PART NO.	QTY.	REMARKS	ITEM

- 1- ELEVATION OF WATER DESIGN LINE FROM SENS. OUTLET TO STANDOFF MUST NOT BE BELOW JACKET WATER OUTLET ON END TO PREVENT TRAPPING OF AIR IN ENG. OR DISCH. LINE
- 2- STANDOFF TO BE SIZED FOR FLOW RATE OF 5 TO 7 FPS. ALLOW SUFFICIENT VOLUME ABOVE WATER LEVEL FOR WATER EXPANSION - APPROX. 50"
- 3- PIPING CONTRACTOR TO PROVIDE HIGH POINT VENTS AND LOW POINT DRAINS AND BLOCK VALVES AS REQD FOR MAINTENANCE
- 4- EXCEPT FOR COPPER AND BRASS FIELD PIPING TO BE CLEANED EGG C-8 STD. 50-64-7 COPPER AND BRASS TO BE CLEANED BY FLUSHING AND SWABING.
- 5- COOLANT SUPPLY TEMP. NOT TO EXCEED 100°F. THE PACKING 200-011 4" STEEL SMALL SIZE FOR A MIN. DRESSURE COOP OR 80 PSI ACROSS THE PACKING CASE.
- 6- COOLANT FLOWS BASED ON 50/50 WATER-GLYCOL SOLUTION.
- 7- PUMP(S) SELECTED ARE BASED ON HISTORICAL USAGE. CUSTOMER TO DETERMINE SYSTEM HEAD MATCH & REVISE C.O.G. IF PROBLEMS EXIST.
- 8- CRANK COMPRESSOR COOLING SYSTEM AS FILLED EXCEPT WHEN GAS TEMPERATURE DIFFERENTIAL EXCEEDS 10°F OR GAS DISCHARGE TEMPERATURE EXCEEDS 250°F. A SLIGHT REDUCTION IN CAPACITY WILL OCCUR WHEN FLOWING COOLANT IS USED.

COLUMBIA GAS TRANSMISSION COMPANY
700 N 150TH AVE
DRAWDG. COMPRESSOR STATION
1951 CUMMINGS, ILL.

BTU

- “BTU” stands for British Thermal Unit
- Equivalent to 778.26 ft-lbf
- Units of ‘work’ are also “ft-lbf”

Horsepower

- A horsepower is equivalent to 33,000 ft-lbf/minute
- Can also be described as “the rate of doing work”

Heating Value of Fuel – Higher and Lower

- “Higher Heating Value” is abbreviated as “HHV”
- HHV assumes that the water content of the exhaust condenses
- “Lower Heating Value” is abbreviated as “LHV”
- LHV assumes that the water content of the exhaust remains in vapor form.
- LHV can be approximated as 91.5% of HHV

Heat Rate Models – Reciprocating Engines

CAT

G3606
G3612
G3616

Cooper-Bessemer

LSV-12
LSV-16
GMV-6TF
GMV-8TF
GMV-10TF
GMV-6STF
GMV-8STF
GMVA-6
GMVA-8
GMVC-8
GMVH-10
GMWA-8
GMWC-8
GMWH-8

Clark

TLA-6
TLA-8
TLA-10
TLAD-10
TRA-6
HSRA-8T

Ingersoll-Rand

36KVS
36KVSRA
410KVG-1
412KVG-1
412KVGB
48KVS
83KVG-1

Superior

6GTLB
8GTLB

Waukesha

3711GU
5108G
7042G
H2475GU

Worthington

ML-10

Heat Rate Models – Gas Turbines

Dresser-Rand

D-R 22 (Allison)

Pratt & Whitney

GG3

GG4

Rolls-Royce

Avon

Solar

Saturn T1002

Saturn T1202

Saturn T1302

Centaur 40 T4002

Centaur 40 T4502

Centaur 40 T4702S

Taurus 60 T7002S

Taurus 60 T7302S

Taurus 60 T7802S

Mars 90 T12002

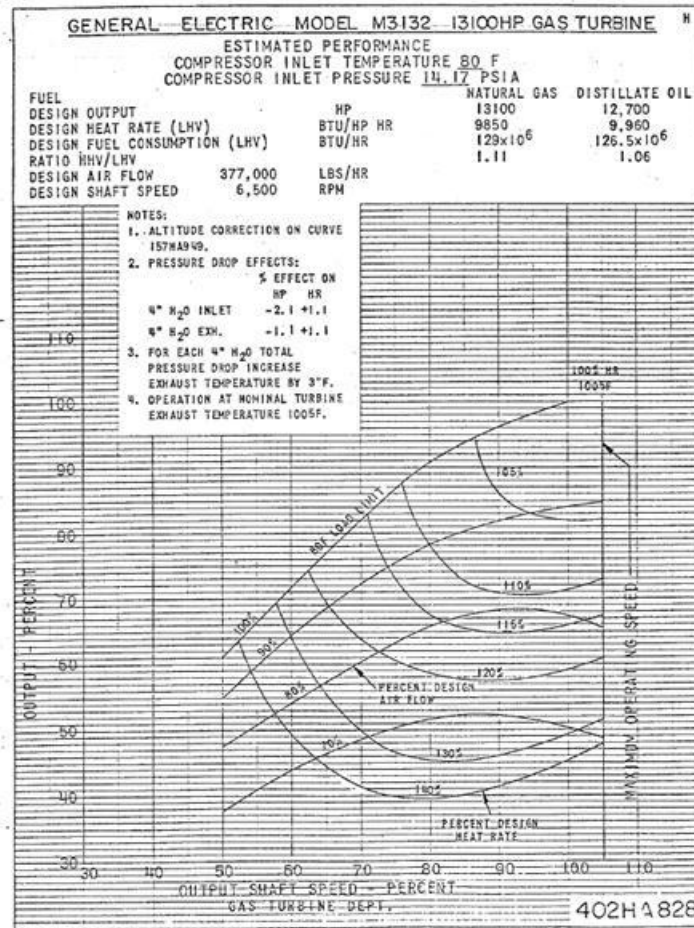
Mars 100 T15002S

Titan 130 T19502S

Titan 130 T19802S

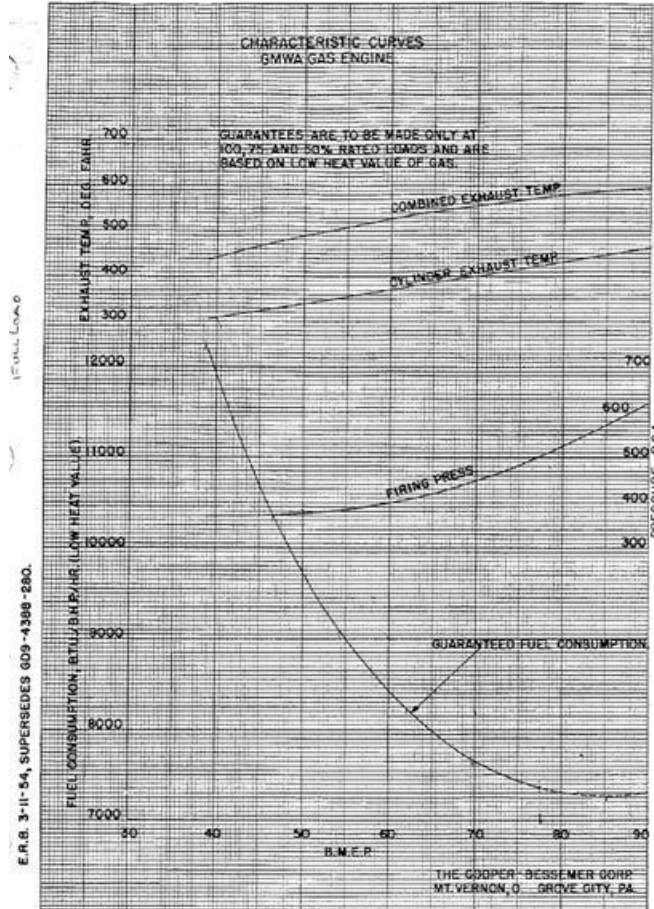
Titan 250 T30002S

Heat Rate – Manufacturer's Info 2-shaft Gas Turbine



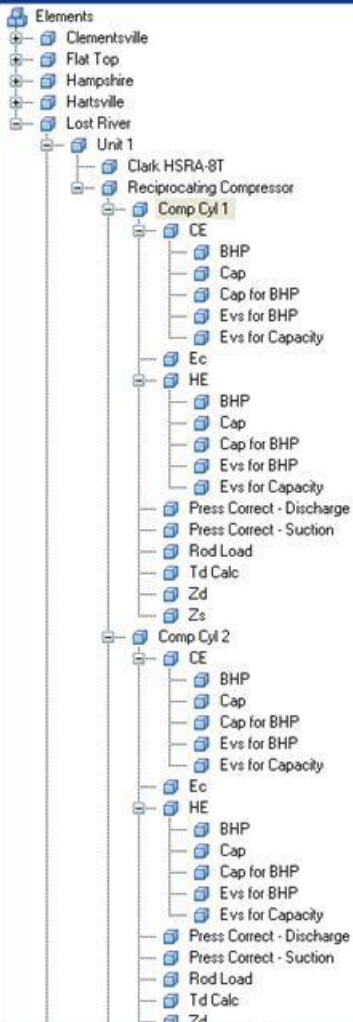
D.F. OSTRANDER JUNE 9, 1970

Heat Rate – Manufacturer's Info Reciprocating Engine



G09-4388-289

Elements



Comp Cyl 1

Comp Cyl 1

Search

Name	Value
CO2 Mole Percent	0
Cylinder Discharge Pressure	719.895385742188
Cylinder Discharge Temperature - Meas	98.8808670043945
Cylinder No	1
Cylinder Suction Pressure	597.389953613281
Cylinder Suction Temperature - Meas	68.1651153564453
Discharge Pressure Correction	0
Ec	0.811408406237817
K	1.3
Mode Desc	
Mode No	0
N2 Mole Percent	0
Palmo	13.8633170862535
Performance Model No	30840
Rc	1.20041681247563
SG	0.59
Station Name	Lost River
Station No	4098
Stroke	14
Suction Pressure Correction	0
Td Calc	96.1734318838899
Unit Discharge Pressure - Panel	719.895385742188
Unit Flowrate	117.138320922852
Unit No	1
Unit Speed	331
Unit Suction Pressure - Panel	597.389953613281
Unloading Method Desc	Piston Displacement / Clearance
Unloading Method No	1
Unloading Step	5
Version No	1
Zd	0.917799787161303
Zs	0.91585525083084

Group by: Category

Name:

Description:

Configuration Item:

Categories:

UOM:

Value Type:

Value:

Data Reference:

Settings...

A=.. \Unit No.[A]

Elements

Transfers

Library

Unit of Measure

Discerning Between ‘Operational’ or ‘Maintenance’ Related Issues

$$\textit{OperationalDifference} = \left(1 - \frac{\textit{ActualHeatRate}}{\textit{TargetHeatRate}} \right)$$

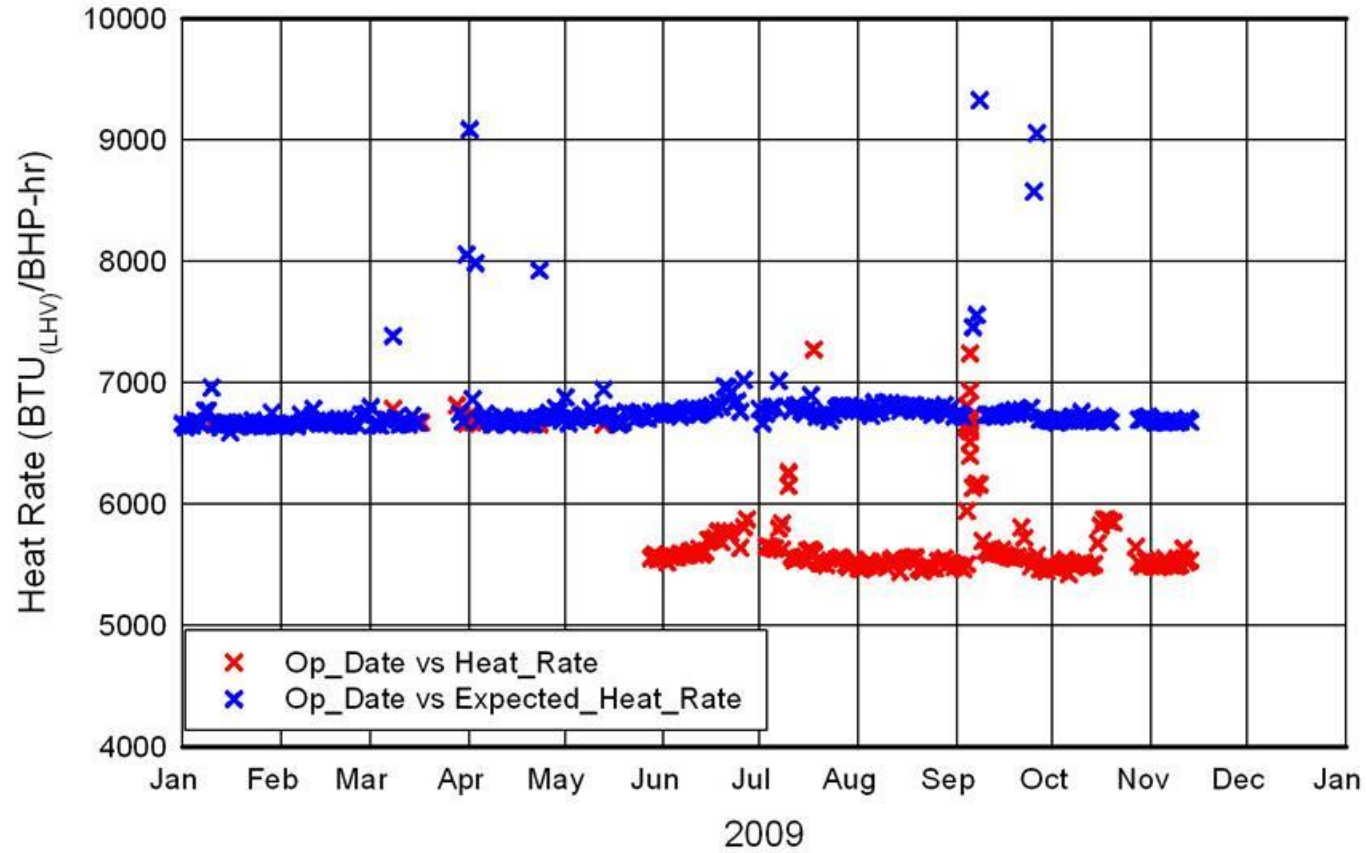
$$\textit{MaintenanceDifference} = \left(1 - \frac{\textit{ActualHeatRate}}{\textit{ExpectedHeatRate}} \right)$$

Heat Rate using Gas Horsepower

$$\textit{EffectiveHeatRate} = \frac{\textit{BTU}_{(LHV)}}{\textit{GasHp} \cdot \textit{hr}}$$

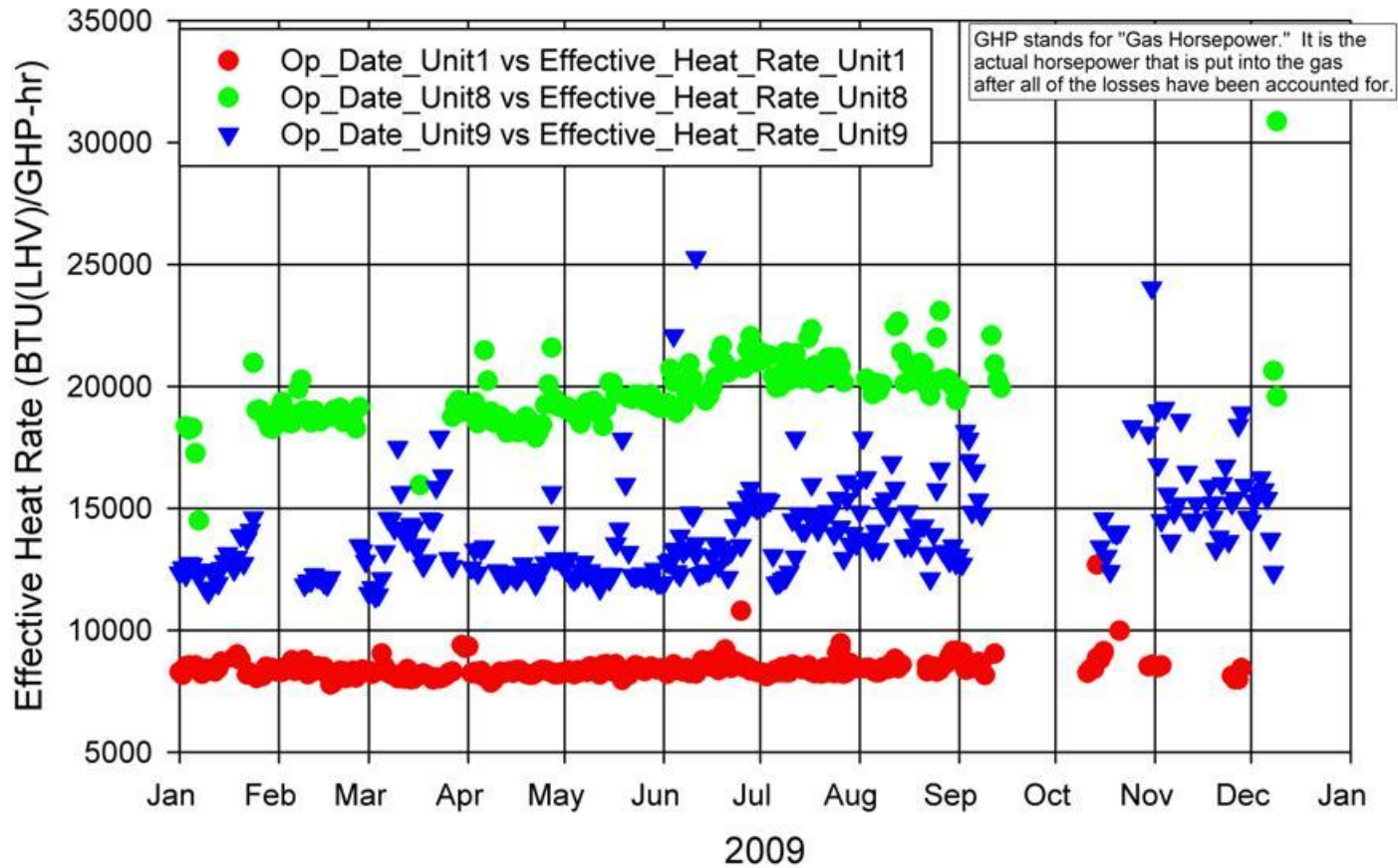
Huff Creek CS Unit #4

Reported versus Expected Daily Heat Rate

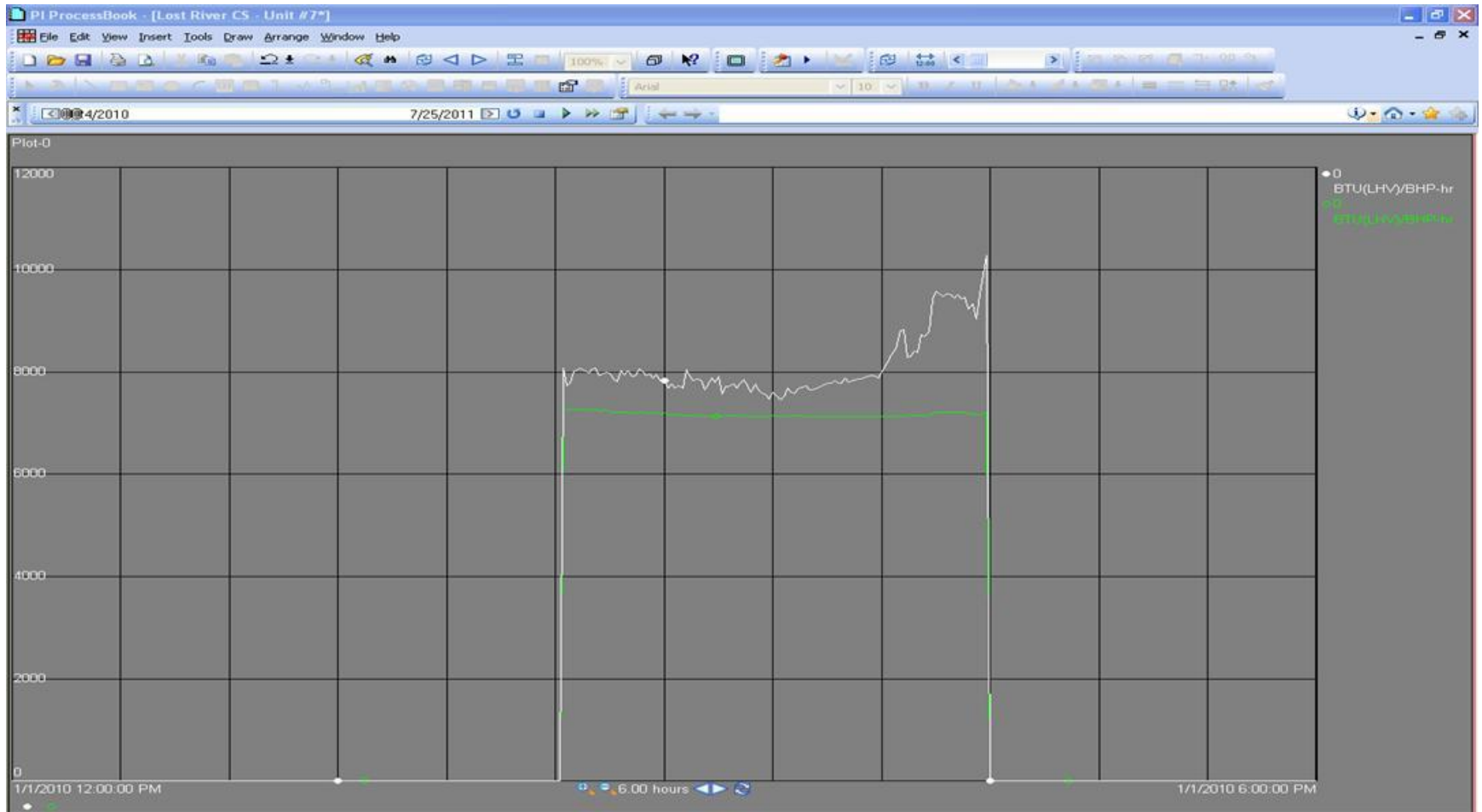


Ceredo CS

Effective Heat Rate Comparison - 2009



Lost River CS Unit #7 – Failure 1/1/2010



Parting Thoughts....

- Compressor calculations whether reciprocating or centrifugal are fairly easy to build into an on-line monitoring system
- Modeling the expected performance of the engine is more difficult
- Before an on-line monitoring tool can indicate a true problem, a lot of work has to be done to verify data inputs and supporting calculations